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On the use of finite elements with a high aspect ratio for modeling cracks in quasi-brittle materials



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ABSTRACT

A new technique for modeling cracks in quasi-brittle materials based on the use of interface solid finite elements is presented. This strategy named mesh fragmentation technique consists in introducing sets of standard low-order solid finite elements with a high aspect ratio in between regular (or bulk) elements of the mesh to fill the very thin gaps left by the mesh fragmentation procedure. The conception of this strategy is supported by the fact that, as the aspect ratio of a standard low-order solid finite element increases, the element strains also increase, approaching the same kinematics as the Continuum Strong Discontinuity Approach. As a consequence, the analyses can be performed integrally in the context of the continuum mechanics, and complex crack patterns can be simulated without the need of tracking algorithms. A tension damage constitutive relation between stresses and strains is proposed to describe crack formation and propagation. This constitutive model is integrated using an implicit-explicit integration scheme to avoid convergence drawbacks, commonly found in problems involving discontinuities. 2D and 3D numerical analyses are performed to show the applicability of the presented technique. Relevant aspects such as the influence of the thickness of the interface elements and mesh objectivity are investigated. The results show that the technique is able to predict satisfactorily the behavior of structural members involving different crack patterns, including multiple cracks, without significant mesh dependency provided that unstructured meshes are used.

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1. Introduction

Although a large number of numerical models are available in literature for modeling cracks in quasi-brittle materials (e.g. concrete and rocks), each one with their own advantages and limitations, this topic is still a challenge, mainly when numerical simulations of multiple cracks in 3D analysis is desirable.

In the context of the Finite Element Method (FEM), the available models can be usually classified into the "smeared crack model" or the "discrete crack model". Both approaches were introduced for modeling cracks in concrete in the late 1960s, the



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Nomeno	lature
n. s. t	components of the local Cartesian cordinate system
n	unit vector – normal to the interface FE
h	height (thickness) of the interface FE
В	strain-displacement matrix
t	interface stresses
d	nodal displacement vector
d, Ĩ	scalar damage variables
f_t	material tensile strength
A	area of the base of the tetrahedron FE
q, r	stress and strain-like internal variables
G_f	fracture energy
$\mathbf{x}^{(i)}$	vector that collects the coordinates of the node (i)
$x_n^{(i)}, x_s^{(i)},$	$x_t^{(i)}$ coordinates of the node (i)
$u_n^{(i)}, u_s^{(i)},$	$u_t^{(i)}$ displacement components of the node (i)
[u]	relative displacement between the node (1) and its projection on the element base $(1')$
$\llbracket u \rrbracket_n, \llbracket u$	$\ _{s}, \ u\ _{t}$ components of the relative displacement
C	fourth order elastic tensor
$\tilde{\mathbb{C}}^{tan}$	algorithmic tangent operator
Ε	Young's modulus
v	Poisson's ratio
3	strain tensor
Ĩ, Ê	components of the strain tensor
$\sigma, \bar{\sigma}$	nominal and effective stress tensors
$\sigma_{nn}, \bar{\sigma}_{nn}$	nominal and effective stress components normal to the base of the interface FE
ϕ, ϕ	damage criterion functions
τ	equivalent effective stress
A	softening parameter

former by Rashid [1] and the latter by Ngo and Scordelis [2]. Basic characteristics of both models and a comparison between them can be found in [3–5].

In the smeared crack model, a fixed Finite Element (FE) mesh is used and the effects of the cracks are incorporated into the constitutive model (stress–strain relation) adopted to describe the material, which is usually nonlinear with a softening regime after the failure (maximum) criterion is reached. This approach is also known as a continuum type representation and the crack zone is considered to be distributed in a certain strain localization (continuum) region of the solid. Models of this class can be formulated using fixed or rotating crack approaches [6–8,4,9]. It is known that the main drawback of standard smeared crack formulations is the mesh sensitivity. To remedy problems of mesh size dependency presented by the models of this class, several regularization procedures have been proposed [10]. Some research papers [11,12] have proposed fracture mechanics concepts that lead to fracture energy release regularization. However, the loss of objectivity associated with the deformation pattern is still observed. Other strategies for dealing with the mesh dependency problem have been addressed using non-local models [13,14] and gradient-enhanced approaches [15].

Discrete models, also referred to as discontinuous models by some authors, are characterized by the introduction of displacement or strain discontinuities into standard finite elements to represent cracks. In general, these methods can be divided into two groups, defined by the location of the displacement discontinuities. Hence, one group gathers the approaches where cracks are embedded into existing bulk finite elements [16–20], and in the other group are the approaches with interface elements in which the cracks are expected between standard bulk finite elements [21–26]. A combination of the two groups is proposed by Radulovic et al. [27]. Models of the former group are usually derived using a nodal-based formulation (e.g. eXtended Finite Element Method – X-FEM) [20,28–30] or element-based formulation (e.g. Strong Discontinuity Approach – SDA) [31,17,32,18,33,34]. A comparative study of these approaches is presented by Oliver et al. [35]. Both methods require techniques to track the crack path during the analysis in order to provide information about the position of the crack surfaces for the discontinuous kinematic enrichment. These techniques are relatively simple to represent few cracks in 2D analyses, but can be very complex and even unsuitable for problems involving multiple crack surfaces in 3D analyses [36–38]. In the X-FEM, the level set method introduced by Osher and Sethian [39] has been used to track cracks.

The necessity of crack tracking schemes can be avoided by using discrete approaches based on the use of interface elements (with a zero thickness, e.g. [21–23,40–44]). In this approach the kinematics of a crack is described by a displacement discontinuity (crack opening width) and a cohesive law is used to describe the material degradation. The use of this kind of approach allows complex crack patterns (crack branching, intersecting cracks and fragmentation) to be simulated automatically. In this paper an alternative discrete approach based on the use of interface solid finite elements for modeling the formation and propagation of cracks is presented. These elements with a high aspect ratio combined with a continuum damage model to describe their behavior was introduced by Manzoli et al. [45]. In this work the authors show that, as the aspect ratio of a standard three-node triangular element increases (ratio of the largest to the smallest dimension), the element's strain also increases, approaching the same kinematics as the Continuum Strong Discontinuity Approach (CSDA) [46,47]. As a consequence, a standard continuum constitutive model, which tends toward a discrete constitutive relation as the aspect ratio increases, can be applied to describe the interface behavior. Details about the equivalence between the strain field of the interface elements and the strong discontinuity kinematics are also demonstrated in Manzoli et al. [45]. The important concept that is used from this equivalence is that bounded stress can be obtained from unbounded strains. Recently, this approach was successfully used in different applications, as for example: modeling of formation and propagation of 3D cracks networks induced by shrinkage in drying soils [48] and 2D modeling of bond slip behavior of reinforcing bars embedded in concrete [49].

Hence, the present approach consists in introducing standard low-order finite elements with high aspect ratio in between all the regular (bulk) finite elements (fragmentation procedure). These interface elements define all potential crack paths, and their behavior is described by a continuum damage model.

Compared with available models based on the introduction of special interface elements (e.g. cohesive zone models and zero-thickness interface elements), the proposed approach presents some advantages, such as:

- It is not necessary to develop a discrete constitutive relation (stress-displacement relation), nor special integration rules to obtain the internal forces, as required by zero-thickness interface elements.
- The interfaces can be modeled entirely in the continuum framework, composed of standard finite elements usually available in FEM codes and continuum constitutive models.
- The use of very high stiffness to reduce the artificial compliance during the elastic regime (normally required in zero-thickness interface elements) is not necessary, avoiding numerical instabilities.
- The proposed discontinuous approach can become available in existing nonlinear FEM codes, practically just by performing a few changes in the pre-processing phase.

This paper is organized as follows. In Section 2 the idea behind the proposed mesh fragmentation technique is presented. The formulation of the interface solid finite elements for 2D and 3D cases is detailed in Section 3. In Section 4 the tension damage model used to describe the behavior of the interface elements is described. In Section 5 the proposed mesh fragmentation technique is assessed in 2D and 3D case studies. Finally, some concluding remarks are given in Section 6.

2. Overview of the proposed mesh fragmentation technique

The proposed technique, hereafter called *mesh fragmentation technique*, is based on the use of interface solid finite elements with a high aspect ratio (proposed by Manzoli et al. [45]) which are inserted in between standard (bulk) finite elements of a finite element mesh. In this work, these interface elements are responsible for describing the crack formation and propagation via an appropriate continuum damage model.

Fig. 1 illustrates the main steps of the proposed mesh fragmentation technique for 2D and 3D problems, which can be summarized as follows:

- 1. Generation of the standard FE mesh to be fragmented (Fig. 1(a)).
- 2. Separation of the finite elements by introducing gaps in between them (Fig. 1(b)).
- 3. Insertion of interface elements with a high aspect ratio in between the bulk finite elements (Fig. 1(c)).

It is important to note in Fig. 1(b) that the gaps are in exaggerated scale for illustration. These gaps are usually very small, with a thickness around 1% of the typical size of the regular elements. The choice of the GAPs thickness is still not a closed issue. However, based on the authors' experience, the assumption of 1% of the typical size of the regular elements seems to be a very reasonable recommendation, provided that the size of the regular elements has been chosen to accurately capture the (elastic) stress field prior to the crack formation. Of course, smaller regular elements produce more accurate responses. The thickness is limited only by the computer precision. In addition to that, triangular/tetrahedron finite elements are used for both the standard FE mesh and the interface elements introduced during the fragmentation process (see Fig. 1).

With the proposed strategy, cracks can only propagate along the interface elements. A reasonable mesh refinement is necessary to capture the failure process. For problems in which the region where cracks are expected is known a priori, the mesh fragmentation technique may be applied only in this region of interest. This approach is very attractive since the steps mentioned above are straightforward in implementation, giving place to an additional pre-processing stage.

The mesh fragmentation approach is completed by a continuum tension damage model formulated to describe the formation and propagation of cracks along the interfaces. As is well known, the constitutive tangent operator may become singular in the strain softening regime, and, as a consequence, the solution of the resulting systems of nonlinear equations using a fully implicit scheme may not be achieved. According to Oliver et al. [50] this problem may also be present in the numerical O.L. Manzoli et al. / Engineering Fracture Mechanics 153 (2016) 151-170



Fig. 1. 2D and 3D mesh fragmentation process: (a) generation of the standard FE mesh to be fragmented; (b) separation of the finite elements (with an exaggerated scale factor for clarity); (c) insertion of interface elements (depicted in gray); and (d) detail of interface elements between regular elements.

modeling of material failure even when a powerful continuation method to pass structural points is used (e.g. arc length methods to transverse limit and turning points). To address this problem, the implicit–explicit (IMPL–EX) integration scheme proposed by Oliver et al. [50,51] is used in this work for the integration of the stress–strain relation of the damage model.

3. Interface solid finite element

To describe the main features of the interface solid finite elements in 2D and 3D modeling, standard three-node triangular finite element and four-node tetrahedron ones, as illustrated in Fig. 2, are considered. The geometry of these elements can be characterized by the position of their nodes according to a local Cartesian coordinate system (n, s, t), defining the unit vector, **n**, normal to the element base, and the height, *h*, given by the distance between the node (1) and its projection on the element base (1').

Following the standard finite element approximations, the strain field of the solid finite element can be expressed by:

(1)

(2)

$$\boldsymbol{\varepsilon} = \mathbf{B}\mathbf{d},$$

where \mathbf{B} is the strain-displacement matrix and \mathbf{d} is the nodal displacement vector of the element.

3.1. Decomposition of the strain tensor

To show the similarity between the kinematics provided by these elements, when h tends to zero, and that associated to the CSDA, the strain tensor is divided into two parts [45]:

 $\mathbf{\epsilon} = \tilde{\mathbf{\epsilon}} + \hat{\mathbf{\epsilon}},$

where $\hat{\varepsilon}$ collects all the components of the strain tensor that depends on *h*, and $\tilde{\varepsilon}$ contains the rest of the components. Thus, the components that depend on *h* can be written as:



Fig. 2. Interface solid finite elements: (a) three-node triangular element and (b) four-node tetrahedron element.

$$\hat{\boldsymbol{\varepsilon}} = \frac{1}{h} (\mathbf{n} \otimes \llbracket \mathbf{u} \rrbracket)^{s}, \tag{3}$$

where $(\bullet)^s$ corresponds to the symmetric part of (\bullet) , \otimes denotes a dyadic product, and $\llbracket \mathbf{u} \rrbracket$ is a vector that collects the components of the relative displacement between the node (1) and its projection on the element base (1').

The total strain tensor, given by Eq. (2), can then be rewritten as:

$$\boldsymbol{\varepsilon} = \tilde{\boldsymbol{\varepsilon}} + \underbrace{\frac{1}{h} (\mathbf{n} \otimes \llbracket \mathbf{u} \rrbracket)^{s}}_{\tilde{\boldsymbol{\varepsilon}}}.$$
(4)

As can be noted from this decomposition, when h tends to zero, the component $\tilde{\epsilon}$ remains bounded, while the $\hat{\epsilon}$ component is no longer bounded. As a consequence, in this situation, the element strains are related almost exclusively to the relative displacement between the node (1) and its projection on the element base (1'). As described by Manzoli et al. [45], in the limit situation ($h \rightarrow 0$), the relative displacement [**u**] becomes the measure of a displacement discontinuity (strong discontinuity), and the structure of the strain field in Eq. (4) corresponds to the typical kinematics of the CSDA. Therefore, based on the same concepts of the CSDA, it can be stated that bounded stress can be obtained from unbounded strains by means of a continuum constitutive relation. The equivalence between the strain field of the interface finite elements (when $h \rightarrow 0$) and the strong discontinuity kinematics is detailed by Manzoli et al. [45]. In this work a continuum tension damage model is used to describe the dissipation effects associated to crack opening (see Section 4).

3.1.1. Components of the strain tensor for the 3-node triangular finite element

According to the local Cartesian coordinate system (n, s), depicted in Fig. 2(a), the nodal coordinates of the 3-node triangular finite element are:

$$\mathbf{x}^{(1)} = (h, b_2),$$

$$\mathbf{x}^{(2)} = (0, 0),$$

$$\mathbf{x}^{(3)} = (0, b).$$
(5)

Thus, the components of the strain tensor of the element, given by Eq. (4), can be expressed as:

$$\tilde{\boldsymbol{\varepsilon}} = \frac{1}{b} \begin{bmatrix} 0 & \frac{1}{2} (u_n^{(3)} - u_n^{(2)}) \\ \frac{1}{2} (u_n^{(3)} - u_n^{(2)}) & (u_s^{(3)} - u_s^{(2)}) \end{bmatrix},\tag{6}$$

and

$$\hat{\boldsymbol{\varepsilon}} = \frac{1}{h} \begin{bmatrix} \boldsymbol{\mathbb{I}} \boldsymbol{\mathbb{I}} \boldsymbol{\mathbb{I}}_n & \frac{1}{2} \boldsymbol{\mathbb{I}} \boldsymbol{\mathbb{I}} \boldsymbol{\mathbb{I}}_s \\ \frac{1}{2} \boldsymbol{\mathbb{I}} \boldsymbol{\mathbb{I}} \boldsymbol{\mathbb{I}}_s & \boldsymbol{0} \end{bmatrix},\tag{7}$$

where $u_n^{(i)}$ and $u_s^{(i)}$ are the components of the displacement of node *i* according to the (n, s) system; while $[\![u]\!]_n = u_n^{(1)} - u_n^{(1')}$ and $[\![u]\!]_s = u_s^{(1)} - u_s^{(1')}$ are the components of the relative displacement $[\![u]\!]_s$.

3.1.2. Components of the strain tensor for the 4-node tetrahedral finite element

In the same way, the nodal coordinates of the 4-node tetrahedron finite element (see Fig. 2(b)) according to a local Cartesian coordinate system (n, s, t) are:

$$\begin{split} \mathbf{x}^{(1)} &= (h, x_s^{(1)}, x_t^{(1)}), \\ \mathbf{x}^{(2)} &= (0, x_s^{(2)}, x_t^{(2)}), \\ \mathbf{x}^{(3)} &= (0, x_s^{(3)}, x_t^{(3)}), \\ \mathbf{x}^{(4)} &= (0, x_s^{(4)}, x_t^{(4)}). \end{split}$$

Thus, the corresponding parts of the strain tensor, given by Eq. (4), can be expressed as:

$$\tilde{\boldsymbol{\varepsilon}} = \frac{1}{A} \begin{bmatrix} \widetilde{\varepsilon}_{nn} & \widetilde{\varepsilon}_{ns} & \widetilde{\varepsilon}_{nt} \\ \widetilde{\varepsilon}_{ns} & \widetilde{\varepsilon}_{ss} & \widetilde{\varepsilon}_{st} \\ \widetilde{\varepsilon}_{nt} & \widetilde{\varepsilon}_{st} & \widetilde{\varepsilon}_{tt} \end{bmatrix},$$
(8)

and

$$\hat{\boldsymbol{\varepsilon}} = \frac{1}{h} \begin{bmatrix} \|\boldsymbol{u}\|_{n} & \frac{1}{2} \|\boldsymbol{u}\|_{s} & \frac{1}{2} \|\boldsymbol{u}\|_{t} \\ \frac{1}{2} \|\boldsymbol{u}\|_{s} & \boldsymbol{0} & \boldsymbol{0} \\ \frac{1}{2} \|\boldsymbol{u}\|_{t} & \boldsymbol{0} & \boldsymbol{0} \end{bmatrix},$$
(9)

where

$$\begin{split} \widetilde{\epsilon}_{nn} &= \mathbf{0}, \\ \widetilde{\epsilon}_{ss} &= \frac{1}{2} \Big[(x_t^{(3)} - x_t^{(2)}) u_s^{(4)} + (x_t^{(2)} - x_t^{(4)}) u_s^{(3)} + (x_t^{(4)} - x_t^{(3)}) u_s^{(2)} \Big], \\ \widetilde{\epsilon}_{tt} &= -\frac{1}{2} \Big[(x_s^{(3)} - x_s^{(2)}) u_t^{(4)} + (x_s^{(2)} - x_s^{(4)}) u_t^{(3)} + (x_s^{(4)} - x_s^{(3)}) u_t^{(2)} \Big], \\ \widetilde{\epsilon}_{ns} &= \frac{1}{4} \Big[(x_t^{(3)} - x_t^{(2)}) u_n^{(4)} + (x_t^{(2)} - x_t^{(4)}) u_n^{(3)} + (x_t^{(4)} - x_t^{(3)}) u_n^{(2)} \Big], \\ \widetilde{\epsilon}_{nt} &= -\frac{1}{4} \Big[(x_s^{(3)} - x_s^{(2)}) u_n^{(4)} + (x_s^{(2)} - x_s^{(4)}) u_n^{(3)} + (x_s^{(4)} - x_s^{(3)}) u_n^{(2)} \Big], \\ \widetilde{\epsilon}_{st} &= \frac{1}{4} \Big[(x_t^{(3)} - x_t^{(2)}) u_t^{(4)} + (x_t^{(2)} - x_s^{(4)}) u_n^{(3)} + (x_t^{(4)} - x_t^{(3)}) u_n^{(2)} \Big], \\ \widetilde{\epsilon}_{st} &= \frac{1}{4} \Big[(x_t^{(3)} - x_t^{(2)}) u_t^{(4)} + (x_t^{(2)} - x_t^{(4)}) u_t^{(3)} + (x_t^{(4)} - x_t^{(3)}) u_t^{(2)} \\ &\quad - (x_s^{(3)} - x_s^{(2)}) u_s^{(4)} - (x_s^{(2)} - x_s^{(4)}) u_s^{(3)} - (x_s^{(4)} - x_s^{(3)}) u_s^{(2)} \Big], \end{split}$$

and *A* is the area of the element base with unit vector **n**; $u_n^{(i)}$, $u_s^{(i)}$ and $u_t^{(i)}$ are the components of the displacement of node *i* according to the (n, s, t) system; $[\![u]\!]_n = u_n^{(1)} - u_n^{(1')}$, $[\![u]\!]_s = u_s^{(1)} - u_s^{(1')}$ and $[\![u]\!]_t = u_t^{(1)} - u_t^{(1')}$ are the components of the relative displacement $[\![u]\!]$ between node (1) and the point corresponding to its projection on the element base (1') $(\mathbf{x}^{(1')} = (\mathbf{0}, \mathbf{x}_s^{(1)}, \mathbf{x}_t^{(1)}))$.

4. Tension damage model

It is well know that constitutive models based on the Continuum Damage Mechanics Theory (CDMT) are appropriate to describe the material degradation process due to crack propagation [52–55]. For this class of models, the mechanical behavior of a damaged material is usually described by using the notion of the effective stress, together with the hypothesis of mechanical equivalence between the damage and the undamaged material.

In this work the phenomenon of crack initiation and propagation through the interface solid finite elements is described by a tension damage model. In the following, the main ingredients of this model and the integration of the stress–strain relation using an implicit–explicit integration scheme are detailed. The resulting discrete relation obtained when the interface element height, *h*, tends to zero is also presented.

4.1. Continuum model

The tension damage model is defined by the following constitutive relation:

$$\boldsymbol{\sigma} = (1-d)\underbrace{\mathbb{C}:\boldsymbol{\varepsilon}}_{\boldsymbol{\sigma}},\tag{10}$$

where σ is the nominal stress; $d \in [0, 1]$ is the scalar damage variable; \mathbb{C} is the fourth order elastic tensor; ε is the strain tensor; and the product $\mathbb{C} : \varepsilon$ defines the effective stress tensor $\bar{\sigma}$.

The damage criterion defines the elastic domain and is given by:

$$\phi = \sigma_{nn} - q(r) \leqslant 0, \tag{11}$$

where σ_{nn} is the component of the stress normal to the base of the element ($\sigma_{nn} = \mathbf{n} \cdot \boldsymbol{\sigma} \cdot \mathbf{n}$), q and r are the stress and strainlike internal variables, respectively, and the function q(r) defines the softening law.

Dividing Eq. (11) by (1 - d), the damage criterion can be expressed in terms of the effective stresses as:

$$\phi = \bar{\sigma}_{nn} - r \leqslant 0, \tag{12}$$

with r = q/(1 - d), which controls the size of the elastic domain in the space of the effective stress. Thus, a damage evolution rule can be defined in terms of the internal variable *r*:

$$d(r) = 1 - \frac{q(r)}{r}.$$
(13)

The constitutive model is completed by the loading–unloading conditions, given by the Kuhn–Tucker relations:

$$\bar{\phi} \leqslant 0, \ \dot{r} \geqslant 0, \ \dot{r}\bar{\phi} = 0, \tag{14}$$

and the consistency condition

$$\dot{r}\bar{\phi} = 0$$
 if $\bar{\phi} = 0$. (15)

Eqs. (15) and (12) lead to the following explicit evolution law for the strain-like internal variable along pseudo-time t associated with the loading process:

157 (16)

$$r_t = \max[\bar{\sigma}_{nn}(s), r_0].$$

According to Eq. (16), *r* takes the maximum value that the effective tensile stress $\bar{\sigma}_{nn}$ reaches during the loading process, starting from the initial value, r_0 , equal to the material tensile strength f_t .

4.1.1. Implicit-explicit integration scheme for the tension damage model

In order to avoid numerical convergence problems related to the strain softening regime, an implicit–explicit (IMPL–EX) integration scheme proposed by Oliver et al. [50] is used for the integration of the stress–strain relation. This method has been widely tested for the integration of constitutive models based on damage mechanics [51] and plasticity theory [51,56].

The IMPL-EX scheme is characterized by the guarantee of convergence (robustness), regardless of the length of the load step, and low computational costs, when compared with fully implicit schemes. For the IMPL-EX scheme only one iteration per load-step is sufficient to establish equilibrium (given by the difference between external and internal forces). As a consequence, there is an error associated with the use of this alternative integration scheme, which can be reduced (or controlled) by decreasing the load step length, as can be seen in the numerous examples performed by Oliver et al. [51].

Table 1 illustrates the steps of the IMPL-EX integration scheme applied to the tension damage model at a pseudo-time step t_{n+1} . The integration procedure is performed in a closed form. For a given strain tensor ε_{n+1} , a corresponding stress tensor $\tilde{\sigma}_{n+1}$ is evaluated explicitly and used to fulfill the balance equation and to compute the effective algorithmic tangent operator. Note that the strain-like internal variable increment evaluated at the previous step t_n is used to explicitly evaluate the approximation of the current strain-like internal variable, $\tilde{r}_{(n+1)}$ (see step (iv) in Table 1), which in turn is used to compute the current damage variable and then the stress tensor. Finally, it is also important to note in Table 1 that the corresponding algorithmic tangent operator is always positive definite, which makes the IMPL-EX methodology very appealing for this type of application.

4.2. Discrete relation of the interface solid finite element

The tension damage constitutive (stress-strain) relation is used to describe the behavior of the interface elements (see previous section). Hence, using Eqs. (10) and (4), the following relation can be written:

$$\boldsymbol{\sigma} = (1-d)\mathbb{C} : \left[\tilde{\boldsymbol{\varepsilon}} + \frac{1}{\underline{h}} \underbrace{(\mathbf{n} \otimes \llbracket \mathbf{u} \rrbracket)^s}_{\hat{\boldsymbol{\varepsilon}}} \right] = \frac{(1-d)}{h} \mathbb{C} : \begin{bmatrix} h \tilde{\boldsymbol{\varepsilon}} + (\mathbf{n} \otimes \llbracket \mathbf{u} \rrbracket)^s \end{bmatrix}.$$
(17)

However, in the limit state of the strong discontinuity, the continuum constitutive relation (Eq. (17)) becomes the following discrete relation between the interface stresses and the components of the displacement jump:

$$\mathbf{t} = \mathbf{n} \cdot \boldsymbol{\sigma} = (1 - d) \hat{\mathbf{K}} \llbracket \mathbf{u} \rrbracket, \tag{18}$$

 Table 1

 IMPL-EX integration scheme for the tension damage model.

INPUT: $\boldsymbol{\varepsilon}_{(n+1)}, \ \bar{\sigma}_{nn_{(n)}}, \ r_{(n)}, \ \Delta r_{(n)}$
(i) Compute the effective stress tensor
$ar{oldsymbol{\sigma}}_{(n+1)} = \mathbb{C}: oldsymbol{arepsilon}_{(n+1)}$
(ii) Compute loading-unloading conditions
if $\bar{\sigma}_{nn_{(n+1)}} \leqslant r_{(n)}$, then
keep the damage threshold: $r_{(n+1)} = r_{(n)}$
else
update the damage threshold: $r_{(n+1)} = \bar{\sigma}_{nn_{(n+1)}}$
(iii) Compute the strain-like internal variable increment
$\Delta r_{(n+1)} = r_{(n+1)} - r_{(n)}$
(iv) Compute the explicit linear extrapolation of the strain-like internal variable
$\tilde{r}_{(n+1)} = r_{(n)} + \frac{\Delta r_{(n)}}{\Delta t_{(n)}} \Delta t_{(n+1)}; \Delta t_{(n+1)} = t_{(n+1)} - t_{(n)} \text{ and } \Delta t_{(n)} = t_{(n)} - t_{(n-1)}$
(v) Update the damage variable
$ ilde{d}_{(n+1)} = 1 - rac{q(ar{r}_{(n+1)})}{ar{r}_{(n+1)}}$
(vi) Compute the stress tensor
$\tilde{\boldsymbol{\sigma}}_{(n+1)} = \begin{cases} (1-\tilde{d}_{(n+1)}) \bar{\boldsymbol{\sigma}}_{(n+1)} & \text{if } \bar{\sigma}_{nn_{(n)}} > 0 \\ \bar{\boldsymbol{\sigma}}_{(n+1)} & \text{if } \bar{\sigma}_{nn_{(n)}} < 0 \end{cases}$
OUTPUT: $\tilde{\sigma}_{(n+1)}, \ \bar{\sigma}_{nn_{(n+1)}}, \ r_{(n+1)}, \ \Delta r_{(n+1)}$
Compute the algorithmic tangent operator
$\tilde{\mathbb{C}}_{(n+1)}^{tan} = \frac{\partial \tilde{\sigma}_{(n+1)}}{\partial \tilde{\sigma}_{(n+1)}} = \begin{cases} (1 - \tilde{d}_{(n+1)})\mathbb{C} & \text{if } \bar{\sigma}_{nn_{(n)}} > 0 \\ \mathbb{C} & \text{if } \bar{\sigma}_{nn_{(n)}} < 0 \end{cases}$

where each component of this relation for 2D and 3D interface elements is provided in Table 2.

To maintain the stresses bounded when the height tends to zero $(h \rightarrow 0)$, all components of the displacement jump must tend to zero if d = 0 (elastic regime). On the other hand, the damage variable must tend to 1 in the inelastic regime with $[\mathbf{u}] \neq \mathbf{0}$. Therefore, in the limit situation of $h \rightarrow 0$, the interface element presents a rigid-damage behavior. Note that, before the stresses reach the damage criterion all components of the displacement jump are precluded via penalization of 1/h. This penalization can be seen in the equation of the discrete constitutive model that emerges from the continuum constitutive model. Note that the elastic stiffness of the discrete model (Eq. (18)) is given by the continuum elastic constants (*K* and *G*) divided by *h*, which corresponds to a very high stiffness for very small values of *h*. This provides a rigid behavior, precluding the evolution of the displacement jump, before the stresses reach the damage criterion.

If the Poisson's ratio is null (v = 0), the discrete relation can be expressed in terms of the Young's modulus, E, since

$$4G/3 + K = E$$
 and $G = E/2$. (19)

For a monotonic increase (in mode I) of the normal displacement ($[\![\dot{u}]\!]_n > 0$, $[\![u]\!]_n|_{t=0} = 0$, $[\![u]\!]_s = [\![u]\!]_t = 0$), the evolution of the normal stress becomes:

$$\sigma_{nn}(\llbracket u \rrbracket_n) = (1-d) \frac{1}{h} E\llbracket u \rrbracket_n = \begin{cases} \frac{1}{h} E\llbracket u \rrbracket_n & \text{if } \llbracket u \rrbracket_n \leqslant \llbracket u_0 \rrbracket_n \\ (1-d) \frac{1}{h} E\llbracket u \rrbracket_n = q(r) & \text{if } \llbracket u \rrbracket_n > \llbracket u_0 \rrbracket_n, \end{cases}$$
(20)

with $[\![u_0]\!]_n = hq_0/E$.

Assuming an exponential softening law of the form

$$q(r) = q_0 e^{\omega / h(1 - r/q_0)},$$
(21)

with $q_0 = f_t$, where f_t is the tensile strength of the material and \mathscr{A} is the softening parameter, the mode I fracture energy, G_f , i.e., the energy dissipated in a complete degradation of the interface element in mode I, is given as

$$G_{f} = \int_{0}^{\infty} \sigma_{nn}(\llbracket u \rrbracket_{n}) d\llbracket u \rrbracket_{n} = \int_{0}^{\llbracket u_{0} \rrbracket_{n}} \sigma_{nn}(\llbracket u \rrbracket_{n}) d\llbracket u \rrbracket_{n}$$
(22)

$$+\int_{\llbracket u_n \rrbracket_n}^{\infty} \sigma_{nn}(\llbracket u \rrbracket_n) d\llbracket u \rrbracket_n$$
(23)

$$G_f = (f_t^2 h/2 + f_t^2/\mathscr{A})/E.$$
 (24)

When *h* tends to zero, the fracture energy tends to a non-null value given as

$$G_f = \frac{f_t^2}{\mathscr{A}E},\tag{25}$$

from which one can define the softening parameter in terms of the material properties as

$$\mathscr{A} = \frac{f_t^2}{G_f E}.$$
(26)

Fig. 3 illustrates the typical curve tension *versus* displacement jump (opening) normal to the interface element, which is subjected to a uniaxial tension. Note that this discrete response corresponds to a rigid-damage behavior.

Finally, Table 3 summarizes the equations describing of the continuum tension damage model and its resulting discrete relation that emerges when the height of the interface solid finite element tends to zero $(h \rightarrow 0)$. From the discrete form it is possible to note that this model is able to deal with cracks evolving in mixed mode, since the damage variable degrades all components of the effective stress.

For numerical applications, a very small (but non-null) value of h can be used, corresponding to the *regularized* kinematics of strong discontinuity [46]. The value of h must be small enough to provide a good representation of the resulting discrete constitutive model that emerges from the continuum model.

Table 2

Components of the discrete relation of the interface elements for 2D and 3D cases.

Dimension of the problem	Interface stresses	Displacement jump	Matrix K
2D	$\mathbf{t} = \left\{ egin{array}{c} \sigma_{nn} \ \sigma_{ns} \end{array} ight\}$	$\llbracket \mathbf{u} \rrbracket = \left\{ \begin{bmatrix} u \rrbracket_n \\ \llbracket u \rrbracket_s \end{bmatrix} \right\}$	$\hat{\mathbf{K}} = \frac{1}{\hbar} \begin{bmatrix} \frac{4}{3}G + K & 0\\ 0 & G \end{bmatrix}$
3D	$\mathbf{t} = \left\{ \begin{array}{c} \sigma_{nn} \\ \sigma_{ns} \\ \sigma_{nt} \end{array} \right\}$	$\llbracket \mathbf{u} \rrbracket = \left\{ \begin{array}{c} \llbracket u \rrbracket_n \\ \llbracket u \rrbracket_s \\ \llbracket u \rrbracket_t \end{array} \right\}$	$\hat{\mathbf{K}} = \frac{1}{h} \begin{bmatrix} \frac{4}{3}G + K & 0 & 0\\ 0 & G & 0\\ 0 & 0 & G \end{bmatrix}$



Fig. 3. Resulting discrete relation of the tension damage model.

Table 3

Continuum and discrete constitutive equations for the tension damage model.

	Continuum model	Resulting discrete model
Constitutive relation	$\boldsymbol{\sigma} = (1-d)\bar{\boldsymbol{\sigma}}$	$\mathbf{t} = (1 - d)\hat{\mathbf{K}}^{\dagger} \llbracket \mathbf{u} \rrbracket$
	$ar{oldsymbol{\sigma}} = \mathbb{C}:oldsymbol{\epsilon}$	
Equivalent effective stress	$ar{ au}=ar{\sigma}_{nn}$	$\bar{\tau} = \bar{\sigma}_{nn} = \frac{1}{h} E[\![u]\!]_n$
Damage criterion	$ar{\phi}=ar{ au}-r\leqslant 0$	$ar{\phi}=ar{ au}-r\leqslant 0$
Evolution law for the strain-type variable	$r = \max_{s \in [0,t]} \ [\bar{\tau}(s), r_0]$	$r = \max_{s \in [0,t]} \ [\overline{\tau}(s), r_0]$
Damage evolution	$d(r) = 1 - \frac{q(r)}{r}, \ d \in [0, 1]$	$d(r) = 1 - \frac{q(r)}{r}, \ d \in [0, 1]$
Hardening/softening law	$q(r) = f_t e^{\mathscr{A}h(1-r/f_t)}$	$q(r) = f_t e^{\mathscr{A}h(1-r/f_t)}$
[†] 2D $\rightarrow \hat{\mathbf{K}} = \frac{E}{h} \begin{bmatrix} 1 & 0\\ 0 & 1/2 \end{bmatrix}$ and 3D $\rightarrow \hat{\mathbf{K}} = \frac{E}{h} \begin{bmatrix} 1\\ 0 & 1\\ 0 \end{bmatrix}$	$\begin{bmatrix} 0 & 0 \\ /2 & 0 \\ 0 & 1/2 \end{bmatrix}.$	

5. Numerical examples

In this section, the ability of the proposed fragmentation approach in predicting different crack patterns is assessed. The interface elements are introduced in between all regular (bulk) elements of the entire mesh or, to minimize computational effort and time process, just in part of the mesh corresponding to the region of study. 2D and 3D examples are carried out using three-node triangular and four-node tetrahedral finite elements, respectively. An element thickness of h = 0.01 mm is adopted for all the cases, except when the influence of this variable is being studied. The behavior of the interface elements is described using the tension damage model while the bulk elements are assumed to be linearly elastic. The softening parameter \mathcal{A} of the damage model is evaluated by the Eq. (26).

5.1. 2D case studies

5.1.1. Plate with hole

In this first example, a plane strain analysis of a rectangular plate with a center hole loaded vertically is carried out. For symmetry reasons, only one-fourth of the plate is analyzed, as shown in Fig. 4. This elastic benchmark problem aims to study the response obtained using two different type of meshes: a standard FE mesh and other fragmented with three different thickness h = 0.005, 0.01 and 0.1 mm. For the standard elements, were adopted Young's modulus of E = 30 GPa and Poisson's ratio v = 0.2, while for the interface elements, Young's modulus of E = 30 GPa and Poisson's ratio v = 0.

Fig. 5 shows the vertical normal stress distribution for all the cases studied. As can be seen, the results present good agreement and a fragmented mesh can reproduce the response obtained using a standard FE mesh in elastic regime in a satisfactory manner. The continuity of the displacement field is shown in Fig. 6.

5.1.2. Three-point bending beams

Herein, a series of experimental tests carried out by Le Bellégo et al. [57] on three-point bending beams are numerically analyzed in order to assess the capability of the proposed mesh fragmentation technique to predict crack propagation in mode I. Fig. 7 shows the test setup of the notched mortar beams of different heights D = 80, 160 and 320 mm, of length L = 4D, and thickness b = 40 mm.

The numerical analyses are carried out in plane stress condition. The behavior of the bulk elements is assumed to be linearly elastic with Young's modulus E = 38.5 GPa and Poisson's ratio v = 0.24. The parameters of the interface model are: Young's modulus E = 38.5 GPa, Poisson's ratio v = 0, tensile strength $f_t = 3.6$ MPa and fracture energy $G_f = 0.05$ N/mm.



Fig. 4. Plate with hole. Geometry, boundary conditions and load.



Fig. 5. Vertical normal stress: (a) standard mesh, (b) fragmented mesh with h = 0.005 mm, (c) fragmented mesh with h = 0.01 mm, and (d) fragmented mesh with h = 0.1 mm.

The analyses are performed using three different unstructured meshes, varying their densities at the central region, as shown in Fig. 7(b). The main characteristics of these meshes are listed in Table 4, in which N_{nodes} is the number of nodes, $N_{elem(r)}$ is the number of regular elements, $N_{elem(i)}$ is the number of interface elements and $N_{elem(t)}$ is the total number of elements.

Fig. 8 shows the deformed meshes for the case with D = 160 mm at the end of the analysis, where the crack pattern can be seen. Strains are mostly concentrated in those interface elements located near the central cross section, following a dominant vertical crack propagation (in mode I) from the notch, with a zigzag path imposed by the mesh geometry, which is not aligned with the expected vertical path. To emphasize the crack pattern, Fig. 8(d)–(f) also show the deformed mesh without the interface elements. The experimental vertical crack could be well predicted for all the cases.

The structural responses (curves of load *P versus* displacement *d*) obtained for the different sizes and meshes are shown in Fig. 9. As can be noted, experimental and numerical responses are in good agreement, and the numerical results do not present significant mesh dependency.



Fig. 6. Vertical displacement field: (a) standard mesh, (b) fragmented mesh with h = 0.005 mm, (c) fragmented mesh with h = 0.01 mm, and (d) fragmented mesh with h = 0.1 mm.



Fig. 7. Three-point bending beam: (a) geometry and boundary conditions and (b) the three mesh refinements used.

Table 4			
Characteristics of the Fl	e meshes of the	three-point l	pending beam

FE mesh	N _{nodes}	$N_{elem(r)}$	$N_{elem(i)}$	N _{elem(t)}
mesh A	6861	4747	5264	10,011
mesh B	14,335	7335	12,672	20,007
mesh C	36,748	14,983	34,962	49,945



Fig. 8. Crack growth trajectories for different mesh refinements (D = 160 mm).



Fig. 9. Structural responses obtained with different specimen sizes and meshes.

In this example the influence of the thickness h adopted for the interface solid finite elements is also investigated. This study was carried out for the beam with D = 160 mm and mesh B. As can be seen in Fig. 10, the results obtained using h = 0.1, 0.01, and 0.005 mm are very similar and could predict well the experimental results.



Fig. 10. Structural responses for interface elements with different thickness (h in mm).



Fig. 11. Geometry and boundary conditions of the four-point bending beam experimentally tested by Arrea and Ingraffea [58].

5.1.3. Four-point bending beam

In this example the four-point bending beam tested by Arrea and Ingraffea [58] is numerically analyzed. The test setup is illustrated in Fig. 11. A fine fragmented mesh is considered only in the region where the crack is expected to propagate (at central region). The linear elastic material properties adopted for the standard (bulk) finite elements were: Young's modulus of E = 28.8 GPa and Poisson's ratio of v = 0.18; and for the interface elements: Young's modulus of E = 28.8 GPa, Poisson's ratio of v = 0, fracture energy of $G_f = 0.1$ N/mm and tensile strength of $f_t = 2.8$ MPa. Plane stress condition is considered with an out-of-plane thickness of 150 mm.

As illustrated in Fig. 12, three unstructured mesh refinements have been considered in order to study the mesh dependency, ranging from a coarse *mesh A* to a fine *mesh C*. The main characteristics of these meshes are listed in Table 5.

The structural responses in terms of load *P* versus δ (vertical displacement between the points A and B) are depicted in Fig. 13(a), and those ones obtained in terms of load *P* versus *d* (vertical displacement at point B) are depicted in Fig. 13 (b). As can be seen in these figures, the numerical responses are in good agreement with the experimental ones. As in the previous example, no significant mesh dependency is observed.

5.1.4. Notched specimen

Fig. 14(a) shows the FE model used to analyze the notched concrete specimen experimentally tested by Kobayashi et al. [59]. The specimen has a thickness of 50.8 mm and two couples of forces F_1 (wedge load) and F_2 (diagonal compression load) are applied increasingly, as illustrated in Fig. 14(b). For the first loading stage, F_1 and F_2 are applied simultaneously, while in the second one, F_2 is kept constant and F_1 is increased.





mesh C

Table 5

Characteristics of the FE meshes of the four-point bending beam.

FE mesh	N _{nodes}	$N_{elem(r)}$	$N_{elem(i)}$	$N_{elem(t)}$
mesh A	16,718	7972	15,204	23,176
mesh B	50,615	20,800	48,188	68,988
mesh C	115,772	44,491	112,170	156,661



Fig. 13. Structural responses of the four-point bending beam.



Fig. 14. Notched concrete specimen: (a) geometry and loading conditions and (b) loading process.

The behavior of the concrete (bulk) finite elements is described by a linear elastic material model with Young's modulus of E = 31 GPa and Poisson's ratio of v = 0.20. For the interface elements, the tension damage model was adopted with Young's modulus of E = 31 GPa, Poisson's ratio of v = 0, fracture energy of $G_f = 0.1$ N/mm and tensile strength of $f_r = 2.7$ MPa.

In this example a mesh dependency is also studied. Fig. 15 shows the three mesh refinements used, whose characteristics are listed in Table 6. A fine fragmented mesh is adopted only in the region where the occurrence of cracks are expected.

Comparisons between experimental and numerical results in terms of crack trajectories and applied load F_1 versus CMOD (Crack Mouth Opening Displacement) are shown in Fig. 16. Note that these results are in good agreement and do not present significant mesh dependency, as can be seen in Fig. 15.

5.1.5. Double-edge-notched specimen

To demonstrate the ability of the technique to describe the propagation of simultaneous arbitrary cracks evolving in mixed mode, the double-edge-notched specimen submitted to a non-proportional loading, experimentally tested by Nooru-Mohamed [60], has been chosen. Fig. 17 shows the test setup discretized with tree-node triangular finite elements in plane stress condition, with an out-of-plane thickness of 50 mm. In this example the mesh of the whole specimen is fragmented, with exception of the mesh of the rigid steel plate used to impose the loading.



Fig. 15. FE meshes and crack trajectories obtained.

Table 6

Characteristics of the FE meshes of the notched specimen.

FE mesh	N _{nodes}	$N_{elem(r)}$	$N_{elem(i)}$	$N_{elem(t)}$
mesh A	2798	1921	2152	4073
mesh B	11,076	5725	9812	15,537
mesh C	43,827	18,671	41,364	60,035



Fig. 16. Results of the notched concrete specimen: (a) crack trajectories and (b) curves of $F_1(N)$ versus CMOD.

The numerical analyses are performed in two loading intervals. First, a lateral force $P_s = 10$ kN is imposed and then, maintaining this force constant, an increasing vertical displacement U_n is imposed to the plate.

A linear elastic model is assumed to describe the behavior of the standard triangular finite elements with Young's modulus of E = 32 GPa and Poisson's ratio of v = 0.18. The behavior of the interface solid finite elements is described by the tension damage model with Young's modulus of E = 32 GPa, Poisson's ratio of v = 0, fracture energy of $G_f = 0.025$ N/mm and tensile strength of $f_t = 2.5$ MPa.

In order to study the mesh dependency, the analyses are performed using different unstructured meshes: coarse (*mesh A*), medium (*mesh B*) and very fine mesh (*mesh C*), as shown in Fig. 18. The main characteristics of these meshes are listed in Table 7.

Fig. 19 shows the crack patterns and the corresponding structural responses. As can be observed, the numerical results predict well the experimental one, with the exception of the peak load, whose predictions are higher than that obtained experimentally for all the meshes. As can be seen, the results do not present significant mesh dependency.

5.2. 3D case study

5.2.1. Double-edge-notched specimen

In order to verify the capability of the proposed mesh fragmentation technique to simulate crack propagation in 3D analysis, the double-edge-notched specimen presented in the previous example (see Section 5.1.5) are numerically simulated. The same loading process, boundary conditions and material properties of the 2D study are assumed.

Fig. 20 shows the three mesh refinements adopted, and Table 8 lists the characteristics of these meshes.

Fig. 20 illustrates the crack patterns obtained using the three meshes considered. As can be seen, the results do not present significant mesh dependency. Moreover, as obtained in the previous example, the structural responses illustrated in



Fig. 17. Double-edge-notched specimen: geometry and boundary conditions.



Fig. 18. FE meshes and crack trajectories obtained.

Table 7

Characteristics of the FE meshes of the double-edge notched.

FE mesh	N _{nodes}	$N_{elem(r)}$	$N_{elem(i)}$	$N_{elem(t)}$
mesh A	2959	1794	2418	4212
mesh B	8470	3632	2280	5912
mesh C	14,826	14,826	20,324	35,150



Fig. 19. Results of the mixed mode test: (a) crack paths and (b) structural responses.



Fig. 20. Crack growth trajectories for the different 3D mesh refinements adopted.

Table 8	
Characteristics of the 3D FE meshes of the double-edge-notched specimen.	

FE mesh	N _{nodes}	$N_{elem(r)}$	$N_{elem(i)}$	N _{elem(t)}
mesh D	7160	2537	9678	12,215
mesh E	12,974	4517	18,204	22,721
mesh F	43,398	9633	33,765	43,398



Fig. 21. Structural responses of the double-edge-notched specimen: (a) 3D numerical analyses versus experimental response and (b) comparison between 2D and 3D responses.

Fig. 21 show that the numerical results in 2D and 3D analyses are in good agreement, and predict well the experimental result, with the exception of the peak load.

6. Concluding remarks

In this paper a discrete crack model using special interface elements is presented for modeling cracks in quasi-brittle materials. The proposed mesh fragmentation technique is based on the use of interface solid finite elements with a high aspect ratio as proposed by Manzoli et al. [45]. This strategy is supported by the fact that these elements are able to describe the kinematics of strong discontinuities in the context of the CSDA when their thickness tends to zero. Hence, standard low-order solid elements with a high aspect ratio can be used to describe the behavior of interfaces representing potential crack surfaces using a continuum constitutive model. Therefore, it is not necessary to formulate a new (solid) contact element for this mesh fragmentation technique.

A simple continuum damage constitutive model based on the tensile stresses normal to the base of these elements is proposed to describe the complete crack formation and propagation process controlled by a prevalent mode I of fracture. It is shown that in the limit situation of strong discontinuity (when the element aspect ratio tends to infinity) the continuum damage model is equivalent to a discrete (cohesive) rigid-damage model, whose softening law can be related to the material fracture energy. Since the damage variable degrades all components of the effective stress, this model is able to deal with cracks evolving in mixed mode. An implicit–explicit integration scheme is used to integrate this constitutive model. With this strategy all the analyses presented in this paper could be performed without problems of convergence, commonly found in problems involving discontinuities.

The numerical analyses carried out showed that the proposed strategy is able to predict the propagation of cracks without the need of a special scheme to track the crack paths during the analysis. For all the examples, the numerical results are in good agreement with the experimental ones presented by various researchers.

No significant mesh dependency is observed, provided that unstructured meshes with a reasonable refinement are used. The misalignment of the mesh geometry only generates a zigzag crack path very close to the expected (or theoretical) crack surface, without prejudice to the structural responses and crack pattern predictions. Irregularity of the crack surface paths is inherent in this approach and may be incompatible with the theoretical smooth surfaces considered for the evaluation of the fracture energy from experimental tests. This difference may be more pronounced in 3D simulations. Therefore, this aspect should be better studied in order to clarify if the assumed fracture energy parameter of the model must reflect the degree of misalignment of the mesh in order to provide the correct energy dissipation.

Finally, with the proposed technique, different crack patterns can be modeled entirely in a continuum framework involving standard solid finite elements and continuum constitutive models compatible with the (regularized) strong discontinuity regime. Therefore, almost only modifications in the pre-processing stage (to introduce finite elements with high aspect ratio in between regular elements of a mesh generated with a standard mesh generator) and in the constitutive model are necessary to make existing nonlinear FEM codes able to use the proposed technique to simulate the effects of cracks.

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